PI and LQR controllers for Frequency Regulation including Wind Generation

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ABSTRACT

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Keyword: Wind Turbines Load Frequency Control Linear Quadratic Regulator Proportional Integral control Power Systems The increasing use of renewable technologies such as wind turbines in power systems may require the contribution of these new sources into grid ancillary services, such as Load Frequency Control. Hence, this work dealt with the performance comparison of two traditional control structures, PI and LQR, for secondary regulation of Load Frequency Control with the participation of variable-speed wind turbines. For this purpose, the doubly-fed induction generator wind turbine was modeled with additional control loops for emulation of the inertial response of conventional machines for frequency regulation tasks. Performance of proposed strategies was verified through simulation in a benchmark adapted from the WSCC 3 machines 9-bus test system. Results showed overall superior performance for LQR controller, although requiring more strenuous control effort from conventional units than PI control.

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1. INTRODUCTION

Electricity production from renewable energy sources (RES) has been continually growing. This development is taking place in a power system structure designed for conventional power sources, with characteristics such as availability, controllability, and reliability utterly different to those of RES systems [1][2]. Also, the energetic production of RES may fluctuate significantly over time due to some characteristics of natural resources, such as unpredictability, variability, and dependency on the geographic location [3] [4]. In particular, some issues attracting a lot of interest in the technical community are the active power variations and frequency performance in presence of RES [1][3] [5], for systems including solar photovoltaic (PV) panels [6] and mostly for wind turbines (WT) [7].

In power systems, frequency constitutes a parameter indicating the equilibrium between power demanded by load and the energy produced by generation systems [8]. When this relationship is unbalanced, control structures are in place to return system frequency to the right operational values. However, these frequency control strategies have been developed for a power system with almost complete reliance on conventional energy sources, and the penetration of RES may require the participation of these new units in the control tasks [3]. Wind turbines, particularly those of variable-speed with doubly-fed induction (DFIG), constitute one of the most used RES around the world [9] [10]. Hence, several studies have been proposed about control strategies for the active inclusion of DFIG WT in Load Frequency Control loops, and complete reviews can be found in references [7] [11] [12].



Figure 1. LFC scheme for a multi-area (*N* areas) power system, including primary and secondary control loops [1]. The block "Wind Turbine Model" integrates WT to LFC.

One of the most used strategies for DFIG contribution in frequency regulation, is the so-called synthetic inertia method, as explained in the studies [13] and [14]. In this technique, additional control loops are developed for the WT with the goal of emulating frequency response of conventional generators. This alternative was explored in other works where controllers based on the dynamic representations of the DFIG WT were proposed. In [15] a Linear Quadratic Regulator type of controller is designed for the WT, taking as model inputs the reference torque and the reference pitch angle of the turbine. The work of Mohamed *et al.* [16] proposes a Model Predictive Controller for WT integration to frequency regulation, using a simplified model of the DFIG with the quadrature-axis rotor voltage as the model input. However, these studies are not exploring WT penetration in a multi-area scenario for power systems, an increasingly common operational possibility as grid grows in size and RES integration level arises. Also, a performance comparison of some of the proposed strategies over the same scenario would be useful to establish the most suitable control structure for WT contribution in frequency regulation tasks. This paper addresses both of the formerly mentioned issues, comparing the performance of PI-based and LQR-based controllers for DFIG WT integration into Load Frequency Regulation (LFC) structure for a multi-area power system. The simulation is performed in a modified version of the 9-bus WSCC power system [8].

This work is the continuation of the research with preliminary results reported in [17]. The former paper focused on the utilization of the synthetic inertia model for WT integration into LFC of power systems with PI controllers. Our current article presents a more elaborated description of the non-linear state-space realization employed for the modeling of variable speed wind turbines. Moreover, DFIG wind turbine operation includes a pitch-angle control loop. Also, additional control structures are explored with the consideration of LQR controllers for secondary regulation, and a performance comparison discussion versus PI strategies. Current paper is divided as follows: Section 2. describes the LFC structure for power systems. Section 3. deals with the WT modeling and the formulation of PI and LQR controllers. Simulation tests and performance comparisons appear in Section 4.. At last, some conclusions are presented in Section 5..

2. LOAD FREQUENCY CONTROL IN MULTI-AREA POWER SYSTEMS

Frequency regulation can be classified in three main stages according to the nature and timing of the control efforts: *primary* actions proportional to the frequency deviations, *secondary* actions allowing correction of steady-state errors, and *tertiary* actions related with predefined dispatches and some emergency conditions. These three stages constitute the Load Frequency Control (LFC) system [8, 1].

Grid elements must be modeled for the design of LFC controllers. First order models are assumed for the governor and turbine of conventional units, and for the representation of the frequency response char-



Figure 2. Wind turbine model with frequency response and variable wind speed (based on [19]).

acteristic of any control area in the power system. Figure 1 shows the LFC for a N-area power system, where parameters for the *i*-th area are: ΔP_{mki} the change in mechanical power of the generator k, ΔP_{gki} the change in the active power output of generator k, ΔP_L the load perturbation, Δf_i the frequency change, D_i the damping coefficient. H_i the equivalent inertia, ΔP_{cki} the control action of the LFC for the k-th generator, T_{ij} the power exchange coefficient between area *i* and area *j*, ΔP_{tiei} the total change in the power exchanged between area *i* and other areas and Δf_j the change in the frequency of area *j* connected to area *i*. Also, B_i denotes the bias factor for modulation of the error signal in secondary regulation, $K_i(s)$ is the transfer function of the secondary controller and α_i the participation factor of each generator in secondary control.

3. INCLUSION OF VARIABLE SPEED WT IN LFC

This work only considered variable-speed DFIG WT, as they are the best-suited WT for active participation in grid ancillary services [18]. However, WT units with DFIG do not present a natural inertial response to frequency changes [1]. For enabling frequency response capabilities to the DFIG WT, synthetic inertia control strategy [14] was employed. This technique proposes operation of the DFIG WT below the point of maximum power extraction to maintain a reserve of kinetic energy to be used for frequency compensation. The operating point P_o depends on the DFIG angular speed w_r and the so-called operational torque T_o [N m], calculated as indicated in equation 1 for different values of wind speed v. Gain K_{op} is adjusted for the operation of the WT under the curve formed by the points of maximum withdrawable power from wind at each speed.

$$T_{op} = K_{op}v^2. (1)$$

For the electromagnetic component of the DFIG, the simplified model proposed in [13] [19] is used and included in the LFC as the *wind-turbine model* block in Figure 1. This representation, denominated as *synthetic-inertia model*, is a reduced induction-machine model of fourth order and only uses the quantities in *q*-axis, as the *d*-axis is selected as the reference frame. Figure 2 presents both models.

In the scheme of Figure 2, P_{base} is the nominal power of the area, w_r is the angular speed of rotor, n is the quantity of WTs, v_{qr} is the rotor quadrature voltage, i_{qr} is the rotor quadrature current, $iq_{r,r}$ is the reference quadrature current for rotor, PIv_{qr} is the PI controller for v_{qr} , T_e is the electromagnetic torque, T_m is the mechanical torque, β is the blade pitch angle, β_r is the blade pitch angle reference, τ_{β} denotes the time delay of pitch angle actuator, $w_{r,r}$ is the rotor angular speed reference, PIv_{qr} is the PI controller for w_r , v_r is the rated wind speed of WT, K_1 is the proportional action of primary control, K_2 is the proportional action of secondary control, K_w and T_w are the gain and time delay of secondary control loop, J is the inertia moment of WT, and $X_1 X_2$, X_3 , and T_1 are approximately constant values representing some combinations among DFIG internal generator parameters (see [13] and [20] for detailed explanation).

As seen from Figure 2, several control loops are added to the simplified turbine model to emulate the behavior of the different control stages of the LFC structure and to keep the stable operation of the DFIG WT after contribution to frequency regulation. These loops are described as follows:

1. A primary response loop for the DFIG labeled as LFC primary loop in Figure 2. Proportional gain K_1 is

modulating the frequency change rate $\frac{d\omega}{dt}$.

- 2. A secondary response loop marked as *LFC secondary loop* in Figure 2 operating in the same way as the secondary control of LFC. The power delivered by WT is restored to the nominal operating point after a control of frequency disturbances.
- 3. The *pitch-angle controller* loop in Figure 2, tasked with maintaining the angular speed of the WT at nominal operating value for wind speeds equal or over the rated ones. Under the action of pitch control, in case of a frequency disturbance occurring, an additional control loop is required for modulating pitch angle with a gain R_{β} proportional to frequency deviation.

The area deviation frequency signal is filtered (through a filter with gain K_a and time delay T_a , see Figure 2) before being applied to primary and secondary control loops. This work performs a comparison of the mentioned loops for two different secondary controllers in frequency regulation. The following subsections describe the PI and LQR secondary controllers (see $K_i(s)$ block in Figure 1) and their interaction with the WT control scheme.

3.1. Considerations for system with secondary PI controller

Proportional Integral (PI) control constitutes the most used variation of the Proportional Integral Derivative (PID) structure [21]. Starting from a simple Single-Input Single-Output (SISO) loop [21], the transfer function $C_{PI}(s)$ of the PI controller is $C_{PI}(s) = K_p(1 + (T_r s)^{(-1)})$. The term K_p is the Proportional gain, T_r is known as the reset time [21], and the relationship $K_p(T_r)^{(-1)}$ is called the integral gain.

PI controllers constitute the traditional strategy for secondary regulation in LFC system. In this work, additional PI controllers regulate quadrature rotor current (PIi_{qr}) and the pitch angle (PIw_r) , as shown in Figure 2.

3.2. Considerations for LFC system with secondary LQR controller

Criterion-based synthesis of controllers is a design technique driven by the complexity of multivariable systems. A commonly employed set of criteria is formed by cost functions related to quadratic forms of control effort and error signals [21]. For linear case, the so-called Linear Quadratic Regulator expresses the problem as the feasible solving of the dynamic Riccati equation in continuous time, leading to a time-variable state feedback [21].

For this configuration, secondary controller $K_i(s)$ requires a state-space representation of the whole LFC. Equations (2) to (11) describe the complete non-linear state-space model for a multi-area power system with integration of WT to the LFC scheme, adapted from [16]. This representation includes the transferred power between areas ΔP_{tie} as a state, with an additional state equation for WT pitch-angle β (see eq. (8)) as a parameter with high influence in the contribution of WT to LFC [15].

$$\dot{\Delta f} = \frac{\Delta P_m - D\Delta f - \Delta P_L}{2H} - \frac{\Delta P_{tie}}{2\pi} + \left(\frac{X_3 w_r i q_r n - P_{ref}}{P_{base}}\right) \frac{\Delta f}{2H}$$
(2)

$$\Delta \dot{P}_{tie} = 2\pi \sum_{j=1}^{N} T_{i,j} \Delta f + \frac{\Delta P_c}{T_g} - 2\pi v_i \tag{3}$$

$$\Delta \dot{P}_m = \frac{-\Delta P_m}{T_\tau} - \frac{\Delta P_g}{T_g} \tag{4}$$

$$\dot{\Delta P_g} = \frac{-\Delta f}{RT_g} - \frac{\Delta P_g}{T_g} + \frac{\Delta P_c}{T_g}$$
(5)

$$\dot{i}_{qr} = \frac{-i_{qr}}{T_1} + \frac{v_{qr}}{T_1}$$
 (6)

$$\dot{w_r} = \frac{-X_3 i_{qr}}{J} + \frac{T_m}{J} \tag{7}$$





Figure 3. WSCC 9-bus system multi-area partitioning. System parameters can be found in [8].

$$\dot{\beta} = \frac{-\beta}{\tau_{\beta}} + \frac{\beta_{ref} + R_{\beta}\Delta f}{\tau_{\beta}} \tag{8}$$

Mechanical torque T_m is calculated from parameters such as air density, length of turbine blades and WT power coefficient C_p (fraction of available wind power being extracted). Differences with [16] involve the consideration of pitch angle reference β_{ref} as an input, and wind speed v and frequency deviation of neighboring areas Δf_j as outputs. Complete vectors of system inputs U and disturbances W are shown below, with ΔP_L the deviations in demanded-load:

Mechanical torque is calculated dividing equation (9) by the angular rotor speed:

$$P_m = \frac{1}{2}\rho\pi R^2 v^3 C_p. \tag{9}$$

where ρ represents air density, R is the length of the turbine blades and the power coefficient C_p denotes the fraction of available power in the wind that is being harvested. This parameter is a function of the Tip-Speed Ratio (TSR) denoted by $\lambda = \frac{Rw_r}{v}$, and the collective blade pitch β .

$$U^{T} = \begin{bmatrix} v_{qr} & \Delta P_{c} & \beta_{ref} \end{bmatrix}, \quad W^{T} = \begin{bmatrix} \Delta P_{L} & \Delta f_{j} & v \end{bmatrix}$$
(10)

Finally, vector Y presents system outputs in equation (11). The first output is the rotor quadrature current i_{qr} , whose reference is given by $i_{qr,r}$. The second output is the system Area Control Error (ACE), reference signal for the LFC secondary controller ($ACE = \Delta f + \Delta P_{tie}$). The last output is the rotor angular speed, whose reference is defined for a given mechanical torque.

$$y^{T} = \begin{bmatrix} iq_{r} & \beta \Delta f + \Delta P_{tie} & w_{r} \end{bmatrix}.$$
(11)

4. **RESULTS**

4.1. Description of case of study

A slightly modified version of the WSCC 9-bus power system [8] was employed for simulation of the DFIG participation in the LFC for a multi-area power. The modified system parameters are summarized in Table 1. This system was partitioned into three areas, as illustrated in Figure 3. Consider Generator 1 as hydraulic and Generators 2 and 3 as gas units. For the sake of this work, 50% of conventional generation in Area III was replaced by a wind farm. The wind farm was formed by 32 DFIG WT of 2 MW each, whose model parameters are shown in Table 2. Wind speed was simulated from a normally distributed random signal with a period of 50 seconds, mean value of 12.5 m/s and variance of 2.8 m/s. Finally, load disturbances were applied as follows: an increment of 0.06 [p.u] at 30 seconds of operation in area III; a variation of magnitude 0.08 [p.u] at 60 seconds in area II; and another disturbance of 0.01 [p.u] at 90 seconds for area I. The Power Base is set at 100 MVA.

Parameter	Value	Parameter	Value	Parameter	Value
H_1	23.64 s	T_{12}	2.064 p.u.	$ $ R_1	2 p.u.
H_2	6.4 s	T_{13}	6.1191 p.u.	R_2	10 p.u.
H_3	1.505 s	T_{23}	14.4353 p.u.	R_3	7.5019 p.u.
MVA_{nom1}	247.5	D1, D2, D3	0.8	B_1	2.8 s
MVA_{nom2}	192	Tg1, Tg2, Tg3	0.2	B_2	10.8 s
MVA_{nom3}	128	$T_{\tau 1}, T\tau 2, T\tau 3$	0.3	B_3	8.3 s

Table 1. WSCC 9 bus system parameters [8].

Table 2	. Wind-turbine mod	tel simulation	parameters [14].
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Parameter	Value	Parameter	Value
P_{nom}	2 MW	R_s	0.00491 p.u.
V_{nom}	966 V	X_{ls}	0.09273 p.u.
K_1	5000 Nm	X_m	3.96545 p.u.
K_2	$2000 \ Nm$	R_r	0.00552 p.u
T_w	1	X_{lr}	0.1 p.u.
K_a	500	H	4.5 s
T_a	20	J	$506.6059 \ Kgm^2$.

4.2. Tuning of PI controllers

PI controllers for secondary frequency regulation in each area were tuned using the Gradient Descent method, along with PI controllers of rotor angular speed w_r and rotor quadrature voltage v_{qr} . Table 3 presents the parameters for all of them.

Table 3. Parameter values for the different PI controllers in simulation for the case of study.

Controller	Proportional Gain k_P	Integral Gain k_I
PI Area I	0	-0.05
PI Area II	0	-0.05
PI Area III	0	-0.28
PI v_{qr}	0	2.70
PI w_r	7.19	0.53

4.3. Tuning of LQR controller

To calculate the gains of LQR controller with reference tracking, a linearization must be performed in the non-linear state-space model described by equations (2) to (8). This process results in the operating point vectors $U_{op}^T = [27.97\ 0.08\ 9]$ for the inputs and $W_{op}^T = [0.2\ 0\ 12]$ for disturbance signals. Design of LQR controllers for secondary LFC implies the tuning of the positive definite matrices Q_{area} and R_{area} for each area. The adjusted matrix elements are listed in Table 4.

Table 4. Parameter values for the different PI controllers in simulation for the case of study. Matrix dimensions for area III are different due to the presence of wind generation.

Parameter	Value
Q_{area1}	$diag([1, 1, 10^{-1}, 10^{-1}, 10^3])$
Q_{area2}	$diag([1, 1, 10^{-1}, 10^{-1}, 10^3])$
Q_{area3}	$diag([10^2, 10^2, 5, 5, 10^{-2}, 10^{-2}, 10^{-2}, 1, 10^6, 10^2])$
R_{area1}	10^{7}
R_{area2}	10^{2}
R_{area3}	$diag([10^2, 10^6, 10^9])$



Figure 4. Frequency deviation in area II. LQR achieved a reduction of 0.32 Hz over PI response in maximum value.



Figure 5. Frequency deviation in area III. Peak deviation value for PI was 0.17 Hz bigger than LQR.

4.4. Comparison between PI and LQR controlled LFC with WT participation

Simulations were performed on the selected benchmark with models and conditions previously described. Figures 4 and 5 depict frequency deviations for areas II and III respectively, as they present more significant variations than area I. Load disturbance in area III at 30 seconds causes the most notorious effects, not only in the local frequency deviation but also in the other areas as well. This behavior could be attributed to the inertia reduction in region 3 and some latency in the operation of WT control loops: power transferred to area III increases as WT contributions in frequency regulation start. On the other hand, the overall magnitude of the frequency deviations over the total simulation time is smaller for the LQR controller in each area. Fewer variations would mean less stress in the regulation systems, a key factor as RES penetration increases. Also, longer recovery and settling times can be seen for PI controllers at each area, giving the LQR a better overall performance for secondary control design in the studied case.

Figure 6 shows the exchanged powers between area III and the other areas. In this case, LQR reduces the power exchanged with other areas when compared with PI response. However, a continued oscillation in power is observed, due to wind variability causing fluctuations in WT generation. This variation makes area III more sensitive to sudden changes in load, as confirmed when the most significant power deviations appear at the same time as the load disturbances occur. With a load disturbance in area III at 30 seconds, LFC system requires an increase in power transference from the other areas to mitigate frequency fluctuations. However, the exchanged power in area III stabilizes as WT start contributing to frequency regulation. Figure 7 shows the power generated by the wind farm in area III.

Focusing on DFIG WT performance, analysis of the control efforts for both LQR and PI strategies in area III is required. Control actions for the pitch-angle β_{ref} are smaller for LQR than for PI controller, as seen in figure 8. This behavior seems to indicate that WT's are less stressed with LQR controller. However, the total control effort of the secondary control ΔP_c is higher for the LQR than the PI scheme, as shown in Figure 9. LQR is imposing an aggressive control action in the conventional unit of area III, diminishing the stress in WT contributions to frequency regulation. This, in turn, reduces frequency fluctuations due to wind variability for



Figure 6. Inter-area power exchange deviation for area III.



Figure 7. Variations in power generated by the wind farm in area III.



Figure 8. Magnitude of control action for variable β_{ref} in area III



Figure 9. Magnitude of control action for variable ΔP_c in area III.

the LQR in this area.

5. CONCLUSIONS

This work studied the performance of variable-speed wind turbines in LFC structure of power systems. With inertial response emulation methodology, DFIG WT were included in the primary regulation stage of the LFC. Two controllers were compared for secondary regulation in the test system, and Linear Quadratic Regulator presented a better overall performance than the Proportional Integral controller. For every explored case, frequency deviations under LQR strategy were smaller and the settling times of the output variables were also lower than the PI-controlled results. Furthermore, LQR operation diminished control efforts of WT. LQR controller based on the system model and it achieved acceptable performances despite the mandatory requirement for linearization of the state-space representation. An unwanted overshoot in area III frequency appears for both strategies with sudden wind variations. This reaction occurs because the operating point of the system is changing with every value of wind speed. This effect was more notorious for LQR configuration, as the model implemented included wind speed as a disturbance. When the operating point changed, linearization might have lead to the inadequate representation of the nonlinear system. Neither PI nor LQR presented a total disturbance rejection, and wind variability may require the pairing of WT with "continuous" generation to reduce operational uncertainty.

Consideration of additional control schemes for the participation of variable-speed WT in LFC enables the contribution of WT to ancillary tasks. However, the increment of WT in power systems may lead to inertia reduction with the decreasing operation of conventional generation systems. More advancements and studies are needed to get a better performance of the wind units in frequency regulation tasks or expanding their role into secondary control. Finally, the implementation of a transition-band control loop between torque and pitch controllers of WT is suggested. This additional loop would help to avoid overshoots in WT generated power when wind varies from nominal speed.

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